RESEARCH

Investigations of the efects of two typical jet crushing methods on the atomization and dust reduction performance of nozzles

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Abstract

Single-fuid nozzles and dual-fuid nozzles are the two typical jet crushing methods used in spray dust reduction. To distinguish the atomization mechanism of single-fluid and dual-fluid nozzles and improve dust control efficiency at the coal mining faces, the atomization characteristics and dust reduction performance of the two nozzles were quantitatively compared. Results of experiments show that, as water supply pressure increased, the atomization angle of the swirl pressure nozzle reaches a maximum of 62° at 6 MPa and then decreases, but its droplet size shows an opposite trend with a minimum of 41.7 μm. The water supply pressure helps to improve the droplet size and the atomization angle of the internal mixing air– liquid nozzle, while the air supply pressure has a suppressive efect for them. When the water supply pressure is 0.2 MPa and the air supply pressure reaches 0.4 MPa, the nozzle obtains the smallest droplet size which is 10% smaller than the swirl pressure nozzle. Combined with the dust reduction experimental results, when the water consumption at the working surface is not limited, using the swirl pressure nozzle will achieve a better dust reduction efect. However, the internal mixing air–liquid nozzle can achieve better and more economical dust reduction performance in working environments where water consumption is limited.

Keywords Dust control · Spray nozzle · Atomization characteristics · Dust reduction performance

1 Introduction

The large amount of dust generated by the deepening of mechanized coal mining has become one of the main reasons for polluting the underground environment, endangering workers' occupational health and safety production. At present, the dust concentration of China's coal workers has far exceeded the concentration limit set by the National Occupational Exposure Limits for Hazardous Agents in the Workplace, which is regulated by national security supervision. On the one hand, a high concentration of dust may cause serious dust explosion. On the other hand, workers' health and safety will be seriously threatened when they work under high dust concentrations for a long time (Yuan

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[2020](#page-14-0); Wang et al. [2017;](#page-13-0) Zhou et al. [2020](#page-14-1)). Exposure to dust over time can lead to pneumoconiosis, an incurable lung disease. According to the data released by the National Health Commission of China, 15407 new cases of various occupational diseases were reported in China in 2021. Among them, 11809 cases of occupational pneumoconiosis were reported, accounting for 76.66% (Xie et al. [2022](#page-14-2); Cheng et al. [2020\)](#page-13-1). Therefore, dust caused by mechanized digging and mining, especially respiratory dust, is a signifcant concern (Wang, [2022](#page-14-3)). At present, many effective technical means of dust prevention and control technology have been applied to production workplaces, including ventilation, dust collector purifcation, spray dust reduction, and airtight isolation (Wang et al. [2019a,](#page-13-2) [b,](#page-13-3) [c](#page-13-4); Liu et al. [2019a\)](#page-13-5). Spray dust reduction is part of coal mine working faces' most widely used dust control technology measures because of its simple and adaptable devices (Wang et al. [2019b,](#page-13-3) [d,](#page-13-6) [e,](#page-13-7) [f](#page-14-4), [g](#page-14-5)).

The spray dust reduction mainly sprays the liquid particles into the roadway through the nozzles, so the atomization nozzle is the fundamental component of spray dust reduction. According to the atomization form, the atomization nozzle can be mainly divided into single-fuid nozzles and

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dual-fuid nozzles (Engelbert et al. [1995;](#page-13-8) Jiang et al. [2017](#page-13-9)). Single-fuid nozzles are widely used in mine, among which swirl pressure nozzles are the most important type of single-fuid nozzles. Wang and Lefebvre [\(2010](#page-13-10)) experimentally obtained an equation for the thickness of the liquid flm at the outlet of the cyclonic nozzle and this equation can be used to predict the Sauter Mean Diameter (SMD). Based on the assumption of defnite liquid flm rupture thickness, Couto et al. ([2011\)](#page-13-11) established a theoretical equation for the SMD of the swirl pressure nozzle. The calculation results of the established theoretical equation are in general agreement with the existing empirical equations, semi-empirical equations, and experimental data. Based on the VOF method, Zhao et al. [\(2017\)](#page-14-6) applied Fluent software to simulate the internal flow of the swirl pressure nozzle. Through numerical simulation, they obtained the variation law of fow feld characteristics parameters such as pressure feld, density feld, and velocity feld in each section inside the swirl pressure nozzle. The study realized the successful prediction of liquid flm thickness and atomization angle at the nozzle exit.

Based on studies of the swirl pressure nozzle, scholars have put forward the idea of using compressed air to enhance the atomization efect of the nozzle (Jones et al. [2010](#page-13-12)). The air–liquid dual-fuid nozzle has gradually been one of the main ways to treat industrial particles and waste pollution. Lorenzetto ([1977](#page-13-13)) found that the increase in liquid surface tension and viscosity will reduce the quality of atomization. So he proposed that when the liquid's viscosity is low, the higher the air–liquid relative velocity at the nozzle outlet, the smaller the droplets produced. In addition, the air velocity is also an important parameter that determines the efect of atomization. In Liu's experimental research, air pressure has the greatest infuence on atomization performance, followed by the air–liquid ratio. While the water pressure has the least infuence from the point of view of the droplet size distribution of the spray axis (Liu et al. [2013](#page-13-14)). Bai's point of view is the same as Liu's, he thinks the air–liquid ratio can efectively reduce the droplet size of the spray, and the air pressure is the main factor that causes the particle size reduction (Bai et al. [2015\)](#page-13-15). Wang et al. [\(2018](#page-13-16)) tested internal mixing air–liquid nozzles with diferent outlet diameters through a self-designed spray dust reduction experimental platform. The experimental data indicated that nozzles with outlet diameters of 2–3 mm are suitable for extraction sites. Li et al. [\(2012\)](#page-13-17) and others pointed out that the flow pattern of the internal mixing nozzle is mainly determined by the air–liquid mass ratio. At the same time, their experiments show that the average diameter of the droplet decreases as the air–liquid ratio increases, and the optimal atomization pressure is 0.2–0.3 MPa.

In summary, there have been many studies on swirl pressure nozzles and internal mixing air–liquid nozzles.

However, most of the studies on nozzles rely on a single type and lack a transversal comparative analysis of swirl pressure nozzles and internal mixing air–liquid nozzles. In addition, most of the research on dust reduction performance of nozzles is focused on the total dust, while there is less research on the reduction efficiency of fine dust such as respiratory dust. Therefore, this study selected the swirl pressure nozzle and the internal mixing air–liquid nozzle as the object of study, and systematically compared the atomization performance and respirable dust reduction performance of the two through self-designed experimental systems. It can provide a reference basis for the design and selection of nozzles for coal mining working face.

2 Experimental systems and scheme

2.1 Nozzle selection

Based on previous site investigations, the nozzles selected for this study are swirl pressure nozzle and internal mixing air–liquid nozzle. The nozzles are made of stainless steel. The outlet diameter of the nozzles is 1.2 mm. The nozzles selected in the experiment are shown in Fig. [1.](#page-2-0) Specifcally, the swirl pressure nozzles, can only generate one highspeed rotating liquid flow. The rotary core is the main part to achieve the spinning, increasing speed and breaking of the jet. The energy of the liquid breakage mainly comes from the pressure energy of the liquid. The internal mixing air–liquid nozzles, can generate high-speed air fow and high-speed liquid fow respectively and mix them inside the nozzle. Compressed air and high-speed jets are mixed and broken in the mixing chamber after entering the nozzle. The energy of the liquid breakage comes from the pressure energy of the compressed air. Internal mixing air–liquid nozzles can achieve a fner atomization efect at a lower pressure. Due to the diferent internal structure of the two nozzles, the atomisation mechanism difers between the two. Internal mixing air–liquid nozzle crushes the internal water fow with highpressure air, which is signifcantly diferent from the atomisation mechanism of swirl pressure nozzle that uses cyclonic flow to enhance droplet crushing.

2.2 Experimental systems

In the experiment, the Phase Doppler Particle Analysis System (PDPA) and spray atomization device were used to measure atomization characteristic parameters of nozzles, the system is shown in Fig. [2.](#page-2-1) The spray atomization device consists of a pressure pump, air compressor, water pipe, air pipe, pressure gauge, pressure limiting valve, water fow meter, rotameter, and the nozzle.

Fig. 1 Experimental nozzle. **a** Swirl pressure nozzle; **b** Internal mixing air–liquid atomization nozzle

Fig. 2 Nozzle atomization characteristics of the experimental system

The dust reduction experiment system via spraying was designed to simulate the dust production, spraying, and ventilation conditions of industrial workplaces, as shown in Fig. [3](#page-3-0). The system consists of 2 sections of $3.0 \text{ m} \times 1.2 \text{ m} \times 1.2 \text{ m}$ rectangular tunnels which made of tempered glass and rigid skeleton spliced together. The dust reduction experimental system, consisting of a dust dispersion system, difusion fan, blower, simulated tunnel, spray device, dust concentration test device and wind speed test device, etc., can simulate the spray dust reduction environment at the underground working face. A single nozzle was used in the experiment. It is located at the middle position of the section and is sprayed in the middle part of the spray section through the water pipe which is perpendicular to the roadway. The axial distance from the dust collector is 15 m.

Fig. 3 Experimental system of dust spraying and dust reduction

2.3 Atomization properties measurement

The nozzle atomization characteristics parameters mainly include fow rate, atomization angle, droplet velocity, and droplet particle size. In the experiment, an electronic water fow meter and digital pressure gauge were used to assess the water flow rate and water supply pressure, respectively. To measure the atomization angle, a high-performance digital camera was used to photograph the spray feld of the nozzle, and then the calculation of the atomization angle was performed using Image-Pro Plus 6.0 software. PDPA was used to measure the droplet velocity and particle size. It is capable of measuring the particle size and velocity of the fow feld measurement points in real-time. The system composition and measurement principle of PDPA are shown in Fig. [4.](#page-3-1)

To fully investigate the atomization characteristics of the swirl pressure nozzle at diferent pressure, the experimental water pressure (p_{water}) was set to 1–7 MPa with a gradient of 1 MPa. While the air–liquid two-phase spraying requires less p_{water} than single-phase flow spraying, the air supply pressure also directly affects the atomization characteristics of the two-fuid nozzle. Therefore, the range of air supply

Fig. 4 Composition of Phase Doppler Particle Analysis System

pressure (p_{air}) variation was set to 0.2–0.4 MPa, and a total of three water supply pressures (0.10, 0.15, 0.20 MPa) were studied. At the same time, the nozzle axial distance of 25, 30, 35, 40, 45, 50, 55, and 60 cm was selected as the measurement points.

2.4 Dust reduction performance measurement

During the measurement of dust reduction performance of the nozzles, the aerosol generator was used to generate dust. Compressed air provided by an air compressor was used as the conveying power to feed the dust from the entrance to the roadway. Two dust testers were placed in the pre-spray measurement section and post-spray measurement section of the model roadway to sample the dust in the two areas under diferent operating conditions. Two dust measurement points were sampled at the same time with a sampling flow rate of 20 L/min for 30 s. The dust concentration was averaged after four consecutive sampling sessions under each working condition to calculate the respirable dust reduction efficiency.

3 Results and discussion on atomization characteristics

3.1 Atomization angle

Atomization angle refers to the liquid being ejected from the nozzle to form an opening angle, which is called atomization angle for short (Wang et al. [2019d,](#page-13-6) [g;](#page-14-5) Liu et al. [2019a,](#page-13-5) [b](#page-13-18)). It is an important atomization characteristic of nozzles. Increasing the atomization angle of the nozzle increases the cross section area covered by the spray feld, which can reduce the number of nozzles required for the on-site application (Yang et al. [2019](#page-14-7)). The atomization angle of the selected nozzles was measured under various operating conditions. The experimental results are shown in Figs. [5](#page-4-0) and [6.](#page-4-1)

Fig. 6 Atomization angle of the internal mixing air–liquid nozzle under different working conditions: **a** p_{water} of 0.10 MPa; **b** p_{water} of 0.15 MPa; **c** p_{water} of 0.20 MPa

Fig. 5 Atomization angle of the swirl pressure nozzle under different p_{water}

As can be seen in Fig. [5](#page-4-0), the atomization angle of the swirl pressure nozzle tends to increase frst and then decrease with increasing p_{water} . The atomization angle at 1.89 L/min is the smallest, and reaches a maximum of 62° at 6 MPa then drops to 60° at 7 MPa with the increase of water supply pressure. This is due to the increase of the nozzle fow rate which increases with increasing p_{water} . Increasing the cyclonic strength in the nozzle is conducive to increasing the radial velocity of the nozzle exit jet. So the atomization angle increases with the increase in p_{water} . However, when p_{water} exceeds 6.0 MPa, the negative pressure in the center of the swirling fow becomes so strong that the fog fow shrinks towards the center, causing the atomization angle to become smaller.

Figure [6](#page-4-1) shows the atomization angle of the internal mixing air–liquid nozzle under diferent operating conditions. It is found that when p_{water} is determined, the atomization angle of the nozzle decreases with the increase of p_{air} . In contrast, when p_{air} is determined, the atomization angle is positively related to p_{water} . This is because the spray field pattern of the nozzle mainly depends on the air phase. The increase in p_{water} strengthens the mixing chamber water pressure, and will affect the air flow in the mixing chamber, resulting in reduced airfow. Since the air–liquid mixing chamber is a shrinking structure, the water pressure enhances the movement of the mixed airfow along the edge of the fare, which eventually leads to an increase in the turbulence intensity at the nozzle outlet and adds atomization angle. While p_{air} increases, the velocity of the air fow increases signifcantly, causing the air fow to be ejected before the expansion, fnally resulting in a decreased atomization angle. In addition, by comparing the atomization angle data in Figs. [5](#page-4-0) and [6](#page-4-1), the atomization angle of the internal mixing air–liquid nozzle is much smaller than that of the swirl pressure nozzle.

From the above analysis, it is found that the swirl pressure nozzle has a larger atomization angle. Therefore, corresponding to the project site to develop a spray dust reduction program, according to the nozzle performance should be a reasonable selection of nozzles. For industrial sites where dust is widely distributed and requires extensive dust suppression, we can choose a swirl pressure nozzle with a larger atomization angle, and increase the water supply pressure appropriately to save the number of nozzles installed.

3.2 Flow rate

Table [1](#page-5-0) shows the measured water flow rates Q_{water} of the swirl pressure nozzle (Test No. 1) as a function of p_{water} and of the internal air–liquid mixing nozzle (Tests No. 2–4) as a function of p_{water} and p_{air} .

For the swirl pressure nozzle, the trend of Q_{water} with the change of p_{water} is shown in Fig. [7](#page-5-1). From the figure, its growth trend gradually slowed down, such as in 1 MPa, 4 MPa, 7 MPa conditions, the fow rate of 1.92, 4.1, 5.4 L/ min respectively. By ftting the experimental data, it was found that there is a power function relationship between water flow rate and water supply pressure, The experimental data can be ftted with a power function with a coefficient of determination of 0.9965. For the internal mixing air–liquid atomization nozzle, it is unlike the swirl pressure

Fig. 7 Variation of Q_{water} with p_{water} of the swirl pressure nozzle

No.	p_{air} (MPa)	Q(L/min)						
		1 MPa	2 MPa	3 MPa	4 MPa	5 MPa	6 MPa	
	$\boldsymbol{0}$	1.89	2.89	3.57	4.08	4.60	5.06	
No.	p_{air} (MPa)	Q(L/min)						
		0.1 MPa	0.15 MPa	0.2 MPa				
$\overline{2}$	0.2	0.29	0.54	0.78				
3	0.3	0.15	0.38	0.52				
$\overline{4}$	0.4	0.07	0.24	0.39				

Table 1 Q_{water} of the two kinds of nozzles under different operating conditions

Fig. 8 Relationships between Q_{water} , p_{water} and p_{air} of the internal mixing air–liquid nozzle

nozzle. The air–liquid mixing pressure in the air–liquid mixing chamber is a major factor afecting the air and water flow. Figure [8](#page-6-0) shows the trend of Q_{water} under different p_{water} and p_{air} . As can be seen from Fig. [8](#page-6-0), with the increase in p_{air} , the water flow is decreasing. For example, at 0.2 MPa p_{water} , when p_{air} is raised from 0.2 to 0.4 MPa, the water flow decreases from 0.781 to 0.387 L/min, which is reduced by about one-half. Due to structural constraints, the fow rate through the mixing chamber is fxed per unit time. An increase in air pressure therefore leads to an increase in the air fow in the mixing chamber, at this time the water fow in the mixing chamber being suppressed by the higher velocity air flow, although the water pressure is then given. Fits through the experimental data at various p_{air} show highly linear relationships between p_{water} and Q_{water} with coefficients of determination greater than 0.986.

By comparing the experimental data of Q_{water} between the two nozzles, it can be found that the water required by the swirl pressure nozzle is much higher than that of the internal mixing air–liquid nozzle. So in practical application, internal mixing air–liquid nozzle can reduce water consumption.

3.3 Droplet size

Droplet size is an important parameter to evaluate the atomization performance of nozzles. We chose D_{32} (Sauter Mean Diameter) to represent droplet size in this paper because it is one of the most commonly used and informative parameters for characterizing droplet sizes. This is beacuse D_{32} takes into account both the volume and number of droplets, thus incorporating size and quantity information comprehensively and accurately. It avoids the drawbacks of other single parameters like D_{10} (tend to be too small) or D_{30} (can be overly infuenced by large droplets). So it can reduce the errors of using one single parameter. In addition, D_{32} is simpler and more practical to calculate and measure compared to obtaining full droplet size distributions and then calculating other diameters. As a single parameter that provides relatively comprehensive and accurate information, D_{32} has advantages in application and calculation convenience. These properties motivate our choice to use it for characterizing droplet sizes in this work.

Figures [9](#page-6-1) and [10](#page-7-0) show the variation curve of droplet size under diferent conditions of the two nozzles. From the fgures can be seen that when p_{water} and p_{air} is fixed, droplet size increases with the increase of axial distance (*S*). Spraying is a process of converting pressure potential energy into kinetic energy of liquid droplets. Collision or condensation of droplets depends mainly on their impact angle and impact velocity. When operating conditions is certain, as the axial distance (*S*) increases, the kinetic energy of the droplets gradually decreases, resulting in an increased probability of condensation between the droplets. So the droplet size at the far end of the spray feld is larger. Meanwhile, the droplet size of the swirl pressure nozzle is decreasing frst and then increases with the increase of water pressure. This is because that when the droplets are in the spray near-feld, droplets have greater kinetic energy and concentration, so the probability of collision breakage is greater, which will result in a decrease in droplet size with the increase in water supply pressure. However, when the water pressure exceeds 3 MPa, the high initial kinetic energy of the droplets makes the water fow inside the nozzle not completely broken before being sprayed. It is a reason for resulting in a gradual increase in droplet size.

Fig. 9 Variation curve of droplet size with p_{water} of the swirl pressure nozzle

Fig. 10 The variation law of droplet size under diferent conditions of the internal mixing air–liquid nozzle

From the droplet particle size data in Fig. [10](#page-7-0), the slope of the curve decreases with increasing p_{air} at p_{water} of 0.2 MPa, indicating that the droplet size growth rate slows down. When p_{air} is given, with the increase in p_{water} the slope of the curve gradually increases, indicating that increasing p_{water} will promote the condensation between the droplets. This is due to the internal mixing air–liquid nozzle which has the ability to use the strong shear properties of the high-speed air fow to enhance the droplet surface fuctuations in a limited space and promote the breaking of the liquid. After the liquid leaves the nozzle, it is wrapped by high-speed air, which drives the droplets to break up further and produce tiny droplets. Unlike the swirl pressure nozzle using p_{water} on the jet to break, highvelocity air is the main factor afecting the droplet breakup of the internal mixing air–liquid nozzle (Han et al. [2020a,](#page-13-19) [b](#page-13-20)).

Comparing with Figs. [9](#page-6-1) and [10](#page-7-0), we can fnd that the droplet size of the internal mixing air–liquid nozzle is always smaller than the droplet size of the swirl pressure nozzle when p_{air} is maintained at 0.4 MPa. This also shows that internal mixing air–liquid nozzle can get smaller droplet size at lower water pressure. Therefore, using the dual-fuid approach to break jet can achieve smaller droplet size while reducing water consumption.

Figure [11](#page-8-0) shows the volume distribution of droplets at each measurement point of the swirl pressure nozzle under 5 MPa water pressure. The orange line in the fgure represents the cumulative percentage of droplet size, and the blue bar represents the droplet volume frequency. From the histogram in Fig. [11](#page-8-0), it can be observed that as *S* increases, the peak droplet volume frequency keeps shifting to the right, i.e., toward the direction of increasing droplet particle size. This means that the maximum droplet size of the swirl pressure nozzle increases with the increase in *S*. This is due to the fact that the kinetic energy of the droplets is gradually reduced by gravity, which increases the probability of condensation between droplets. This leads to an increase in the maximum droplet size of the nozzle. From the cumulative curves, it can also be found that the characteristic particle sizes of droplets (*D*90, *D*50, and *D*10) all show the same variation pattern as Fig. [9](#page-6-1), i.e., they increase with the increase of axial distance.

The droplet size distribution of the internal mixing air–liquid nozzle at $p_{\text{water}}=0.2$ MPa and $p_{\text{air}}=0.4$ MPa at each measurement point is shown in the Fig. [12](#page-9-0). It can be seen from the figure that in the near-field spray $(40 cm),$ the droplet size volume distribution is "spike-like", and the highest peak value accounted for less than 50 μm particle size interval. While in the far-field spray $(>40 \text{ cm})$, the particle size distribution tends to be dispersed. The peak droplet volume distribution gradually shifted to the right with the increase in the axial distance, and the droplet size in Fig. [10](#page-7-0) shows the same pattern of change. This is due to the internal mixing air–liquid nozzle has a smaller initial droplet size, so even though the droplet velocity in the spray near-feld is larger at this point, more non-centric collisions occur, resulting in a larger droplet size. This causes droplets of small size to become progressively larger as *S* increases and the droplet velocity decreases. The droplet peak gradually shifts to the right and causes the volume distribution to become progressively more dispersed.

3.4 Droplet velocity

Figures [13](#page-10-0) and [14](#page-10-1) are the variation curve of axial droplet velocity with diferent operating conditions for the two nozzles. It can be seen from the fgures that when the water pressure is fxed, the droplet speed decreases as *S* increases. Moreover, the higher the water pressure, the greater efect on the velocity of the near-feld spray. The liquid completed atomization near the outlet to form droplets, then the droplets moved along the nozzle axis at a high initial velocity. Due to the effect of air resistance, the droplet velocity continuously

Fig. 11 Volume distribution of droplets at different measurement points under 5 MPa p_{water}

decayed along the nozzle axis. Combined with the atomization angle experimental results, we can fnd that when the water supply pressure increases, the swirl pressure nozzle has a more dispersed fog flow. Therefore, the spray field downstream droplet velocity decay is more dramatic, resulting in further reduction in droplet velocity. From Fig. [13](#page-10-0) we can see that, when p_{water} exceeds 4 MPa, the effect of water supply pressure on droplet velocity enhancement has been very weak in the spray near field ($S \leq 40$ cm). This may be because the nozzle structure limits the flow coefficient, resulting in little increase in liquid per unit time through the nozzle outlet despite the high water supply pressure. Another reason may be the near-feld droplet concentration of the high-pressure nozzle, which increases the likelihood of droplet collisions.

From Fig. [14](#page-10-1) we can find that when p_{air} is low $(< 0.4$ MPa), the droplet velocity is logarithmically related to the spray distance and the droplet velocity change rate decreases gradually. But when p_{air} reaches 0.4 MPa, the two are linearly related. Combined with the droplet particle size of the internal mixing air–liquid nozzle, the droplet velocity under 0.4 MPa p_{air} is low, but the number of droplets is high. This results in a large number of collisions between droplets consuming energy, thus the droplet velocity decreases exponentially. While in 0.4 MPa p_{air} conditions, Q_{water} of this nozzle becomes smaller, resulting in smaller droplet particle size, higher velocity, and inter-droplet collision is greatly reduced. Therefore, the velocity is linearly decreased.

By comparing Figs. [13](#page-10-0) and [14](#page-10-1), the droplet velocity of the internal mixing air–liquid nozzle at each measurement

Fig. 12 Volume distribution of droplet size at each measurement point under p_{water} 0.2 MPa and p_{air} 0.4 MPa

point is always less than the droplet velocity of the swirl pressure nozzle at high p_{water} ($>$ 3 MPa). However, the difference in maximum droplet velocity between the two nozzles is not much within 10%. The droplet velocity of the internal mixing air–liquid nozzle is influenced by both p_{air} and p_{water} . On the one hand, the water supply and air supply pressure of this nozzle is much smaller than that of the swirl pressure nozzle. On the other hand, the high-speed airfow and water fow which mix in the nozzle cavity break the jet while while weakening the kinetic energy of both. Therefore, at the outlet of the internal mixing air nozzle the droplet velocity is small. Droplet size and droplet velocity are important indicators of nozzle atomization performance. Smaller droplet size and higher droplet velocity facilitate respirable dust deposition (Wang et al. [2020](#page-14-8); Charinpanitkul and Tanthapanichakoon [2011](#page-13-21)). After comparing the droplet size and droplet velocity between the swirl pressure nozzle and internal mixing air–liquid nozzle under diferent operating conditions, we found that the internal mixing air–liquid nozzle can obtain the same droplet velocity as the swirl pressure nozzle by using less water, and its droplet size is smaller. Therefore, it has better atomization performance, especially when the water supply pressure is low.

Fig. 13 Variation curve of droplet velocity with different p_{water} of the swirl pressure nozzle

4 Results and discussion of dust reduction performance

4.1 Dust mass concentration and respirable dust reduction efficiency

Respirable dust is one of the most important substances endangering human health and is also the focus and difficulty in dust hazard prevention and control. For this reason, dust reduction experiments were conducted using dust below 10 μ m to study the dust removal efficiency of two nozzles on respirable dust.

Tables [2](#page-11-0) and [3](#page-12-0) show the dust concentration (c_{mt}) of the measured section before and after spraying of the two nozzles under diferent conditions. From the table, it can be seen

that the dust mass concentration in the measurement section before spraying is close.

Figure [15](#page-11-1) shows the change in dust reduction efficiency (μ_r) of the two nozzles under different operating conditions. As we can see from Fig. [15](#page-11-1)a, for the swirl pressure nozzle, the respirable dust reduction efficiency gradually increases as p_{water} increases. According to the experimental results on the atomization characteristics of the nozzles, the swirl pressure nozzle with high water pressure has a large fow rate, atomization angle, and a low droplet particle size. Therefore, it can efectively increase the density of the droplets after the jet was broken and the collision probability between the droplets and the dust. Finally, it can help improve dust reduction efficiency. Although the droplet size is larger when the water supply pressure reaches 6 MPa, the nozzle has a higher water flow and droplet speed. The dust is washed by highspeed water, resulting in a slight increase in dust reduction efficiency.

From Table [3](#page-12-0) and Fig. [15](#page-11-1)b, we can see that when the p_{water} is 0.1 MPa, the respirable dust reduction efficiency of the internal mixing air–liquid nozzle decreases gradually with the increase of p_{air} , from 40.31% at 0.2 to 31.95% at 0.4 MPa. The spray droplet particle size at this time is very small, and the droplet speed is very high, but the water flow rate under 0.2 MPa p_{air} is only 0.284 L/min, resulting in the number of droplets being very small. When p_{water} exceeds 0.15 MPa, the dust reduction efficiency increases first and then decreases with the increase of p_{air} . This is due to the increase in p_{air} significantly reducing Q_{water} and atomization angle at low p_{water} . Although the spray droplet particle size is very small at this time, the number of droplets and spray field coverage is small, resulting in droplets being difficult to trap dust, thus reducing the efficiency of dust reduction (Yu et al. 2018). Therefore, a certain spray flow rate is a prerequisite to guarantee the efficiency of dust reduction.

Fig. 14 Variation curve of droplet velocity under diferent conditions of the internal mixing air–liquid nozzle

Table 2 Dust reduction efficiency of swirl pressure nozzle for respirable dust at different p_{water}

p_{water} (MPa)	Particulate	$c_{\rm mt}$ (mg/m ³)	$\mu_{\rm r} \left(\% \right)$	
	size (μm)	Before spray- ing	After spraying	
1	$0 - 2.5$	26.60	17.30	34.96
	$2.5 - 10$	11.10	5.50	50.45
	$0 - 10$	37.70	21.80	42.18
$\overline{2}$	$0 - 2.5$	26.80	16.70	37.69
	$2.5 - 10$	11.50	5.40	53.04
	$0 - 10$	38.30	22.10	42.30
3	$0 - 2.5$	25.90	15.50	40.15
	$2.5 - 10$	11.50	4.60	60.00
	$0 - 10$	37.40	20.10	46.26
$\overline{4}$	$0 - 2.5$	27.50	15.40	44.00
	$2.5 - 10$	11.90	5.30	55.46
	$0 - 10$	39.40	20.70	47.46
5	$0 - 2.5$	25.80	14.40	44.19
	$2.5 - 10$	11.40	5.10	55.26
	$0 - 10$	37.20	19.50	47.58
6	$0 - 2.5$	25.50	14.12	44.63
	$2.5 - 10$	13.00	5.76	55.69
	$0 - 10$	38.50	19.88	48.36
7	$0 - 2.5$	26.50	14.46	45.43
	$2.5 - 10$	14.50	6.50	55.17
	$0 - 10$	41.00	20.96	48.88

4.2 Comparative analysis of application conditions of nozzles

According to the results of dust reduction experiments, when p_{water} reaches 4 MPa, the growth of respirable dust reduction efficiency of the swirl pressure nozzle is small. At this time, the atomization angle growth of the swirl pressure nozzle is not large, but the droplet particle size is smaller, and the droplet velocity is greater. Therefore, after considering the economy, energy saving, atomization performance, and dust reduction efficiency, for dust production sites that can realize high-pressure water supply and need a large range of close spraying, we can choose 4 MPa water supply pressure to obtain better dust reduction efficiency. Although there are still defects of large water consumption and serious water pollution.

According to the results of the internal mixing air atomization nozzle dust reduction experiments, it can be found the nozzle dust reduction efficiency difference is not significant in the water pressure 0.15 MPa, air pressure 0.30 MPa, and water pressure 0.20 MPa, air pressure 0.30 MPa. However, when water pressure is 0.150 MPa, the nozzle has a smaller water fow rate and droplet particle size. Although the atomization angle of the internal mixing air–liquid nozzle is always below 20°, it has a further spray distance and requires less water. Therefore, in the dust production sites which inability to install other equipment and needs longdistance dust reduction, we can choose the internal mixing

Fig. 15 Dust reduction efficiency of two nozzles under different conditions. **a** The swirl pressure nozzle; **b** The internal mixing air–liquid nozzles

Table 3 Respiratory dust reduction efficiency of the internal mixing air–liquid nozzle at diferent air–liquid pressures

air atomization nozzle to obtain a better dust reduction efect. It can also help to avoid excessive water caused by water pollution, water waste, and other problems.

5 Conclusions

In this study, the swirl pressure nozzle and the internal mixing air–liquid nozzle were selected as representatives of single-fuid nozzles and dual-fuid nozzles. The atomization characteristics of the selected nozzles were compared through experiments using PDPA System, and flow measurement instruments. On this basis, experiments on the respirable dust reduction performance of nozzles were carried out using a self-designed spray dust reduction experimental platform. The following conclusions were drawn:

(1) The water fow rate of the two nozzles increases with water supply pressure increasing. Meanwhile, the atomization angle of the swirl pressure nozzle reaches a maximum of 62° at 6 MPa and then decreases. For the internal mixing air–liquid nozzle, the atomization angle and the water supply pressure are positively correlated, while the air pressure will inhibit the atomization angle. The maximum atomization angle of this nozzle is only one-third of that of the swirl pressure nozzle.

- (2) The droplet size of the swirl pressure nozzle tends to decrease with a minimum of 41.7 μm and then increase as the water pressure increases, but the droplet velocity is proportional to the water supply pressure. The droplet speed of the internal mixing air–liquid nozzle is affected by both water supply pressure and air supply pressure. When both of the nozzles under a smaller water pressure, the droplet size of the internal mixing air–liquid nozzle is 10% smaller than the droplet size of the swirl pressure nozzle and the speed of the two is similar.
- (3) The dust reduction experimental results show that the dust reduction efficiency of the swirl pressure nozzle increases with an increase in water supply pressure. The dust reduction efficiency of the internal mixing air–liquid nozzle decreases with the increase of air sup-

ply pressure when water supply pressure is low. However, when water supply pressure reaches 0.15 MPa, the dust reduction efficiency shows a trend of increasing frst and decreasing then with the increase of air supply pressure.

(4) The swirl pressure nozzle at a water supply pressure of over 4 MPa achieves a better dust reduction efect. It is appropriate for industrial dust-producing environments that can realize high-pressure water supply. For industrial dust production sites which inability to install other equipment and also need long-distance dust reduction, it is better to use the internal mixing air atomization nozzle to obtain a better dust reduction effect.

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Author contributions HH: Methodology, validation, formal analysis, writing-original draft, visualization. HW: Conceptualization, resources, writing-review and editing, supervision, project administration, funding acquisition. QZ: Methodology, resources, writing-original draft. YD: Validation, investigation, data curation. HW: Validation, data curation. HW: Validation, data curation.

Declarations

Conflict of interest The authors declare that they have no known competing fnancial interests or personal relationships that could have appeared to infuence the work reported in this paper.

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